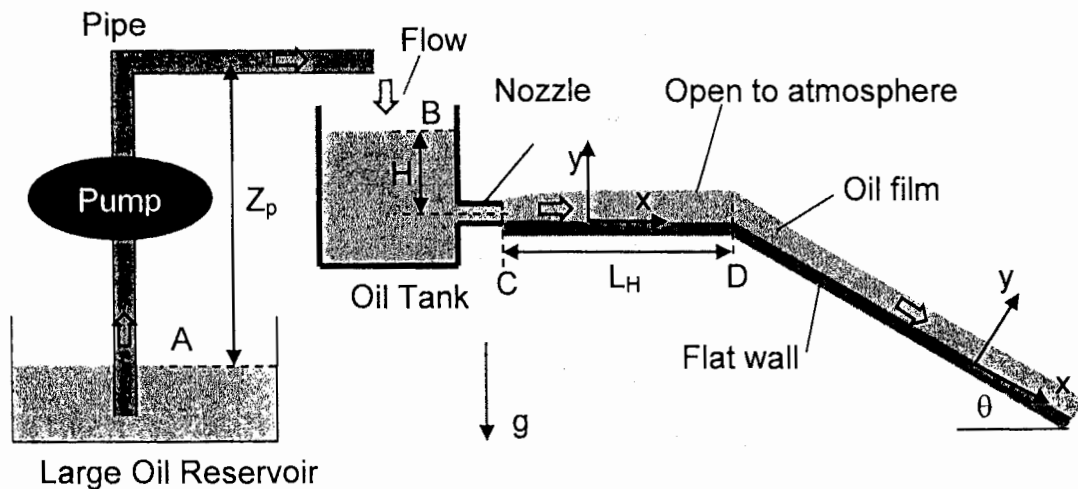


※ 考生請注意：本試題不可使用計算機

1. (50 %) Steady oil flow exists in the system shown in the figure below. The pump delivers enough oil from a large reservoir through the inverted L-shaped pipe to keep the oil level in the oil tank,  $H$ , constant. Total length of the pipe is  $L_p$ , and the diameter is  $d_p$ . The height of the horizontal pipe section from the free surface of the oil reservoir is  $Z_p$ . Oil flows out of the oil tank through a nozzle onto a horizontal flat plate. Assume frictionless flow in the tank up to the nozzle exit. The velocity of the oil flow at point C is therefore constant and equals to  $U$  across the nozzle height,  $c$ . Boundary layer begins to develop in the oil film as soon as the oil film leaves the nozzle, and the boundary layer thickness is defined as  $\delta$ . The oil film thickness at point D equals to  $a$ , and the distance between point C and D is  $L_H$ . The flow goes down an inclined plate at an angle of  $\theta$  with the horizontal direction beyond point D, and becomes fully-developed with a constant thickness of  $h$  after a short adjustment distance. Widths of the plates are large, and both equal to  $b$ . There are side walls so that the oil does not spill from the sides. Density and kinematic viscosity of the oil are  $\rho$  and  $\nu$ , respectively. Specific gravity,  $g$ , is pointing vertically downwards. The directions of  $x$  and  $y$  coordinates are shown in the figure. Show all assumptions, derivations and calculations for the problems below.



- (1) Derive the expression for  $U$  as a function of  $g$  and  $H$ .
- (2) The velocity profile in the boundary layer of the oil film on the horizontal section is given as  $u/U = 2(y/\delta) - (y/\delta)^2$ . Derive the relationship among  $a$ ,  $\delta$ , and  $c$ .
- (3) Find the expression for the drag force on the surface from point C to D by applying control volume analysis.
- (4) If the velocity profile in the fully-developed flow region on the inclined section is  $u/U_{max} = 2(y/h) - (y/h)^2$ , what is the drag force per unit length in this section? Justify each step in the solution and eliminate  $U_{max}$  in the final answer.
- (5) Find an expression for the pump power necessary to keep the oil in the oil tank at a constant height of  $H$  when the flow in the pipe is laminar. Neglect minor losses.
- (6) If the working fluid is replaced with water (kinematic viscosity smaller than oil), how will the boundary layer thickness at point D be affected given  $H$  remains the same, and why?

(背面仍有題目,請繼續作答)

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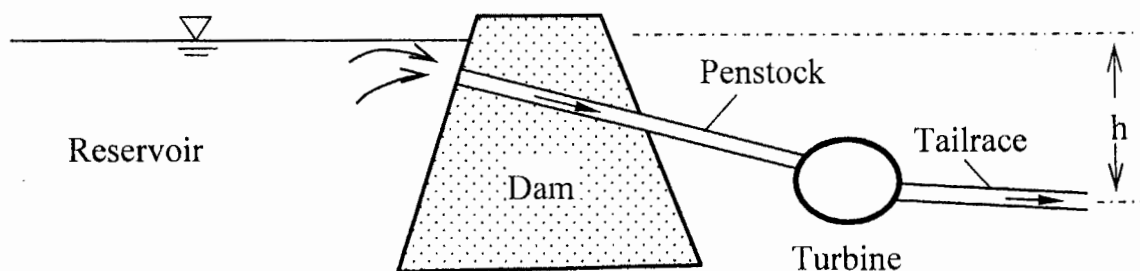
2. (10%) The following is the stream function for a particular steady, planar, incompressible and inviscid flow:

$$\psi = A(x^2y - \frac{y^3}{3})$$

where A is a know constant.

- (1) Find expressions for the velocity components  $u$  and  $v$  in this flow.
- (2) Find an expression for the vorticity.
- (3) We can define a velocity potential for this flow. Why? Find an expression for the velocity potential by assuming that the value of the velocity potential at the origin is zero.
- (4) Make a rough sketch of the streamlines of this flow.
- (5) Find an expression for the pressure in this flow assuming that the pressure,  $p_0$ , at the origin is known. Denote the fluid density by  $\rho$  and neglect all body forces. What shape are the lines of constant pressure (isobars)?

3. (20%) The tailrace (discharge pipe) of a hydro-electric turbine installation is at an elevation,  $h$ , below the water level in the reservoir:

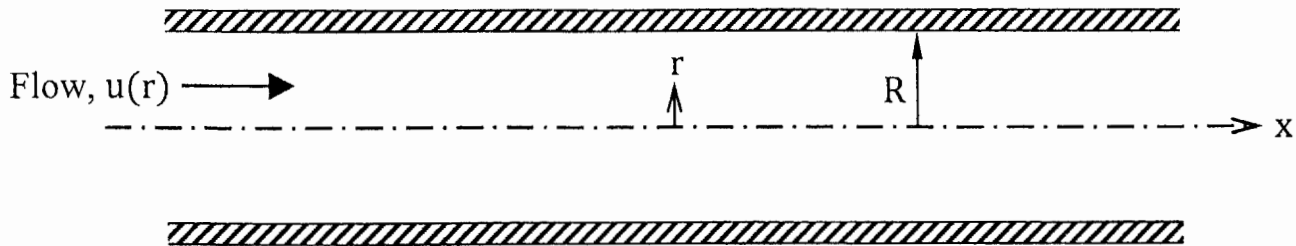


The frictional loss in the penstock (the pipe leading to the turbine) and the tailrace are represented by the loss head,  $kU^2/2g$ , where  $U$  is the mean velocity in those pipes (which have the same cross-sectional area,  $A$ ) and  $g$  is the gravitational acceleration. The flow discharges to atmospheric pressure at the exit from the tailrace. The water density is denoted by  $\rho$ .

- (1) What is the drop in total head across the turbine?
- (2) What is the power developed by the turbine assuming that it is 90% efficient?
- (3) What is the optimum velocity,  $U$ , which will produce the maximum power output from the turbine assuming that  $h$ ,  $k$ ,  $A$ ,  $\rho$  and  $g$  are constant?

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4. (20%) Consider the fully-developed pipe flow of an incompressible, non-Newtonian fluid:



The fluid is such that the normal stress in the  $x$ -direction is equal to  $-p$ , where  $p$  is the pressure and the shear stress,  $\sigma$ , is related to the velocity gradient by

$$\sigma = C \left( -\frac{du}{dr} \right)^2$$

where  $C$  is a known constant. Find the friction factor,  $f$ , for this pipe flow in terms of  $C$ ,  $\rho$  (the fluid density) and  $R$  (the radius of the pipe).

[Note: Remember the definition

$$f = \frac{4R}{\rho \bar{u}^2} \left( -\frac{dp}{dx} \right)$$

where  $\bar{u}$  is the average velocity of the pipe flow.]